

Lecture on the subject
KKE/TSM - Boosting combustion engine theory

doc.Ing. Jiří Polanský Ph.D.



Podpořeno v rámci projektu CZ.1.07/2.2.00/15.0383
Inovace studijního oboru Dopravní a manipulační technika
s ohledem na potřeby trhu práce

Boosting combustion engine theory

Basic parameters

BASIC PARAMETERS

- **Time of one stroke**

- $t_z = \frac{1}{2f} = \frac{60}{2n} [s]$

f – frequency of the revolution [Hz]

n – engine revolutions [min⁻¹]

- **Average piston speed**

- $c_s = \frac{Z}{t_z} = \frac{Zn}{30} [m \cdot s^{-1}]$

Z – stroke [m]

- **Air density (inlet)**

- $\rho_s = \frac{p_s}{r \cdot T_s} [kg \cdot m^{-3}]$

p_s – inlet pressure [Pa]

r – gas constant [J · kg⁻¹ · K⁻¹]

T_s – inlet temperature [K]

- **Compression ration**

- $\varepsilon = \frac{V_1}{V_2} [-]$

V₁ – Volume on the compression start [m³]

V₂ – Volume on the compression end [m³]

BASIC PARAMETERS

- **Fuel mass flow rate**

- $\dot{m}_p = \frac{m_p}{t} [kg \cdot s^{-1}]$

- m_p – fuel amount [kg]

- **Fuel caloric value H_u**

- $H_u = \frac{Q}{m_p} [J \cdot kg^{-1}]$

Q – energy released [J]

- **Specific fuel consumption**

- $m_{pe} = \frac{\dot{m}_p}{P_e} [kg \cdot s^{-1} \cdot W^{-1}]$

P_e – Effective power [W]

- **Middle indicated pressure- p_i**

- $p_i [Pa]$

- Theoretical constant value. Work during one stroke equals work during entire cycle

- **Middle effective pressure- p_e**

- $p_e = p_i \cdot \eta_m [Pa]$

- Indicated pressure multiplied by mechanical efficiency (pressure recalculated to the engine clutch). With mechanical losses

BASIC PARAMETERS

- Specific work – **w**
- Specific work (turbines) – **w_t**
- Ratio of specific volume work – **w_v** and swept volume **V_{z1}**

$$w_V = \frac{W}{V_{z1}} = \frac{\int_{ob\ddot{e}h} p \cdot dV}{V_{z1}} \quad V_{z1} = \frac{\pi \cdot D^2}{4} \cdot Z$$

BASIC PARAMETERS

- Effective power – **P_e** [kW]
- Power of the internal combustion engine :

$$P_e = \frac{i_v \cdot i_p \cdot V_{z1} \cdot p_e \cdot n}{30 \cdot \tau}$$

- V_{z1} – swept volume of one cylinder [dm³]
- p_e – effective middle pressure [MPa]
- i_v – number of cylinders
- i_p – number of active surface
- τ – number of strokes
- V_{z1} · i_v – engine swept volume

BASIC PARAMETERS

- Power – **P**
- Adiabatic cycle: $P = \dot{m} \cdot (h_p - h_0)$
- Power of the internal combustion engine:

$$P = \frac{W_1 \cdot i_v \cdot i_p}{t_1} = \frac{W_1 \cdot 2 \cdot f \cdot i_v \cdot i_p}{\tau} = \frac{W_1 \cdot n \cdot i_v \cdot i_p}{30 \cdot \tau}$$

- W₁ – work of one cycle during time t₁
- f – frequency of revolution of the crankshaft
- n – engine RPM
- i_v – number of cylinders
- i_p – number of active surfaces
- τ – number of strokes

BASIC PARAMETERS

- Torque – **M_M** (W_M=W_e) [kN.m]

$$M_M = \frac{i_v \cdot i_p \cdot V_{z1} \cdot p_e}{\pi \cdot \tau}$$

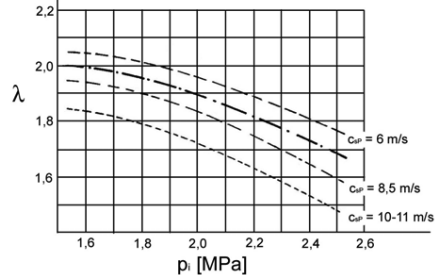
- V_{z1} – swept volume of one cylinder [dm³]
- p_e – effective middle pressure [MPa]
- i_v – number of cylinders
- i_p – number of active surface
- τ – number of strokes
- V_{z1} · i_v – engine swept volume

BASIC PARAMETERS

- The coefficient of excess air in cylinder λ_e .

$$\frac{\rho_s}{\lambda_e} = \rho_s \cdot m_p \cdot \frac{L_t}{V_z \cdot \rho_s}$$

- Nondirective effect of λ_e to p_i



Graph of dependence λ, p_i and c_s [2]

ρ_s – inlet density
 m_p – amount of the fuel

L_t – fuel consumption
 c_s – average piston speed

BASIC PARAMETERS

- Thermodynamic efficiency η_t

$$\eta_t = 1 - \frac{Q_o}{Q_{pc}} [-]$$

Q_o – amount of output heat[J]

Q_{pc} – amount of input heat[J]

- Chemical efficiency η_{ch}

$$\eta_{ch} = \frac{Q_p}{Q_d} [-]$$

Q_p – amount of heat during combustion[J]

Q_d – chemical energy of the fuel (given by caloric value of the mixture) [J]

- Coefficient of indicator diagram - η_p

$$\eta_p = \frac{W_i}{W_t} [-]$$

W_i – indicated mechanical work [J] (work of working media – depends on evolution of the pressure)

W_t – theoretical work of the cycle[J]

- Indicated efficiency η_i

$$\eta_i = \frac{W_i}{Q_d} = \eta_t \cdot \eta_{ch} \cdot \eta_p [-]$$

BASIC PARAMETERS

- Transport efficiency η_d

$$\eta_d = \frac{m_e}{V_z \cdot \rho_s} [-]$$

m_e – mixture amount in cylinder [kg]

V_z – swept volume [m³]

ρ_s – density at inlet [kg.m³]

- Volume efficiency η_o

$$\eta_o = \frac{V_e}{V_z} [-]$$

V_z – clearance volume[m³]

V_e – volume of inlet air [m³]

- Filling efficiency η_{pl}

$$\eta_{pl} = \frac{m_s}{\rho_s \cdot V_z} [-]$$

m_s – amount of media at engine inlet[kg]

V_z – swept volume[m³]

ρ_s – inlet density of the filling media[kg.m³]

- Mechanical efficiency η_m

$$\eta_m = \frac{W_e}{W_i} = \frac{P_e}{P_i} [-]$$

Ratio of effective mechanical work (W_e [kJ]) / effective power (P_e [W]) at engine inlet and indicated (theoretical) mechanical work (W_i [kJ]) / indicated power (P_i [W]) at engine output

BASIC PARAMETERS

- Effective efficiency η_e

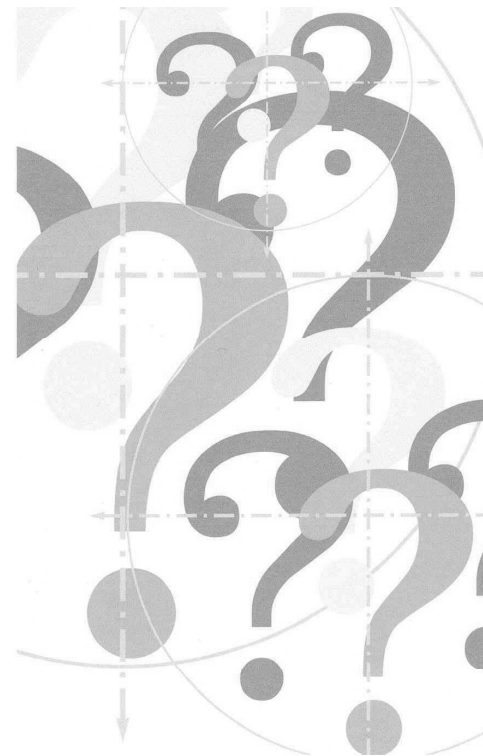
$$\eta_e = \frac{W_e}{Q_d} = \eta_i \cdot \eta_m [-]$$

Ratio of the effective (real) mechanical work (W_e [J]) on the crankshaft output and chemical energy given by fuel (Q_d [J]). It's indicated efficiency multiplied by mechanical efficiency.

References

- [1] J. Macek; B. Suk : Spalovací motory I. - Praha 1996
- [2] L. Bartoníček: Přepřínování pístových spalovacích motorů – Liberec 2004
- [3] K. Hoffman: Regulované přepřínování vozidlových motorů. Brno, 2000.
- [4] J. Macek; V. Kliment: Spalovací turbíny, turbodmychadla a ventilátory (Přepřínování spalovacích motorů) – Praha 2003
- [5] Hiereth H., Prenninger P.: Charging the Internal Combustion Engine, Springer, Wien 2007
- [6] Bell C : Maximum Boost, Bentley Publishers, Cambridge – 1997
- [7] Baines C.N.: Fundamentals of Turbocharging, NREC, Vermont 2005

DISCUSSION...
...QUESTIONS



Poděkování

Tento projekt je spolufinancován
Evropským sociálním fondem a státním rozpočtem České republiky

Projekt CZ.1.07/2.2.00/15.0383
Inovace studijního oboru Dopravní a manipulační technika
s ohledem na potřeby trhu práce