

Lecture on the subject  
KKE/TSM - Boosting combustion engine theory

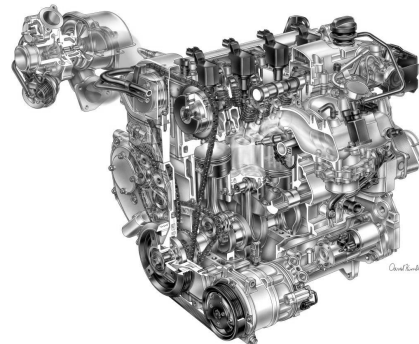
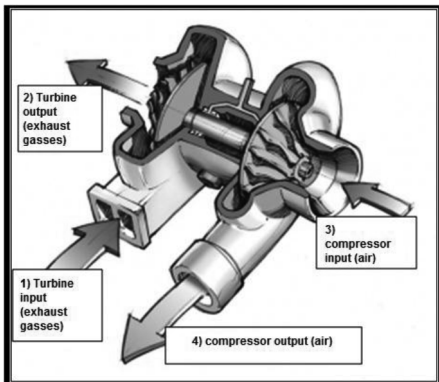
doc.Ing. Jiří Polanský Ph.D.



Podpořeno v rámci projektu CZ.1.07/2.2.00/15.0383  
Inovace studijního oboru Dopravní a manipulační technika  
s ohledem na potřeby trhu práce

## Turbochargers

- Currently (2013) most used method of engine supercharging
  - Lower fuel consumption
  - Robust construction and higher power



*Turbocharger – principle of work [17] [18]*

# Boosting combustion engine theory

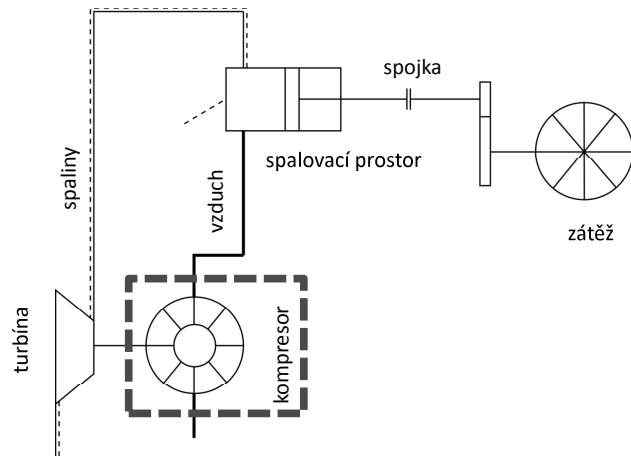
## Compressors

## Turbochargers

### Principle of work

- Exhaust gasses leaving combustion space and moving forward to the turbine section (1). In turbine section exhaust gasses spins turbine impeller and leaving turbine section (2) through the exhaust manifold
- Turbine section is connected with compressor section via shaft
- Turbine section spins the compressor section (compressor's impeller
- According dynamic phenomena created by compressor's impeller, the air flows into compressor (3). After that the compressor compress the air and transport it to the combustion space (4).

## Turbochargers



*Turbocharger – schematic (without intercooler)*

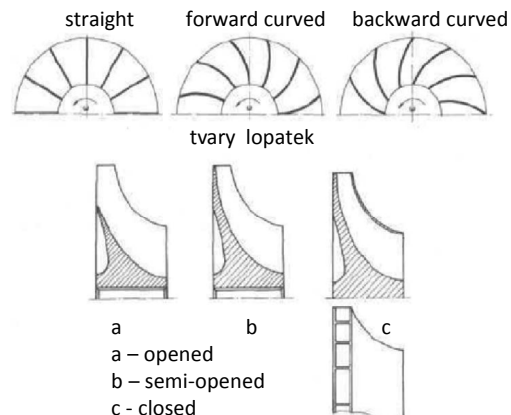
## Centrifugal compressor

- Compressor is a device which provide a compression of air and its transport to combustion chamber
- Compression in centrifugal flow compressor is achieved by centrifugal velocities
- The air in outlet from blade canal is upright to axis of rotation
- Velocity of outlet air depends by diameter of blades and rotor's RPM

## Compressor types

### • Impeller and case shape

- Straight
- Forward curved
- Backward curved
- Opened
- Semi-opened
- Closed



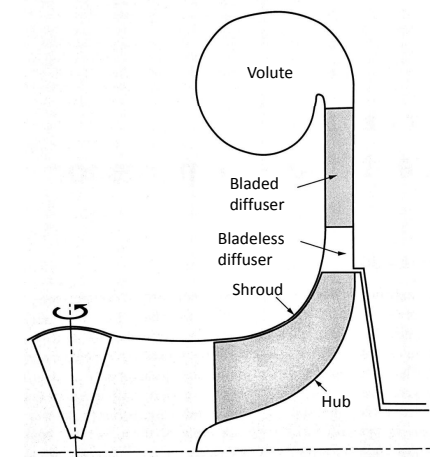
## Main parts

### Rotors

- Impeller
- Hub

### Stator

- Diffuser
  - Bladed
  - Bladeless
- Shroud
- Manifold



*Hlavní části radiálního kompresoru [7]*

# Oběžné kolo

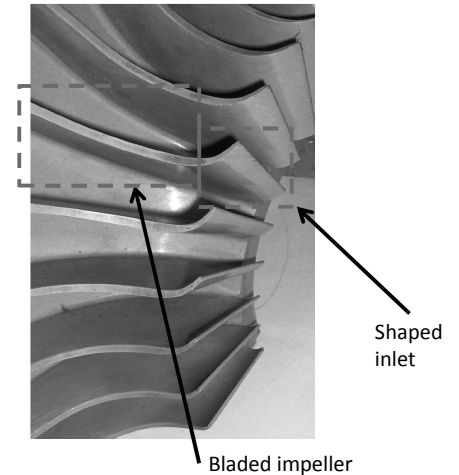
## Oběžné kolo

- Provides transformation of impeller's mechanical energy to air's kinetic and pressure energy
- As a result of centrifugal forces in the compressor channel there are increasing values of absolute velocity and total pressure

# Impeller

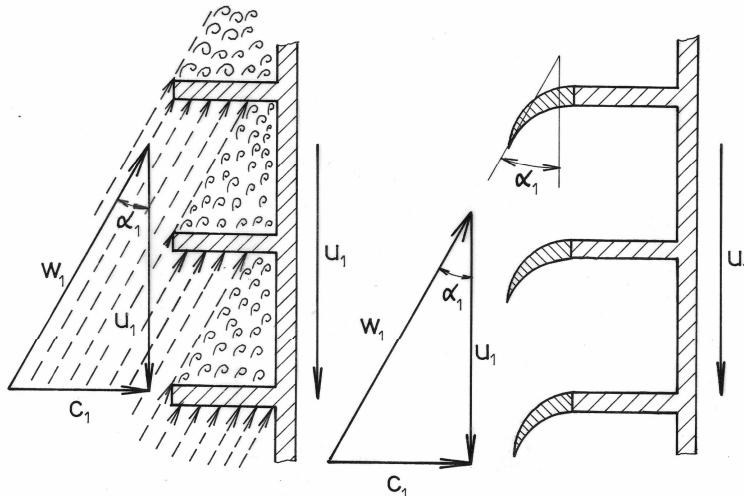
## Impeller – Main parts

- Shaped inlet
- Bladed impeller
- Shaped inlet
  - Curved inlet part of the blades, which provides fluent flow of the air in the compressor inlet.



Detail of the centrifugal compressor impeller

# Impeller



Impact of the shaped inlet to the flow parameters in compressor [7]

# Diffuser

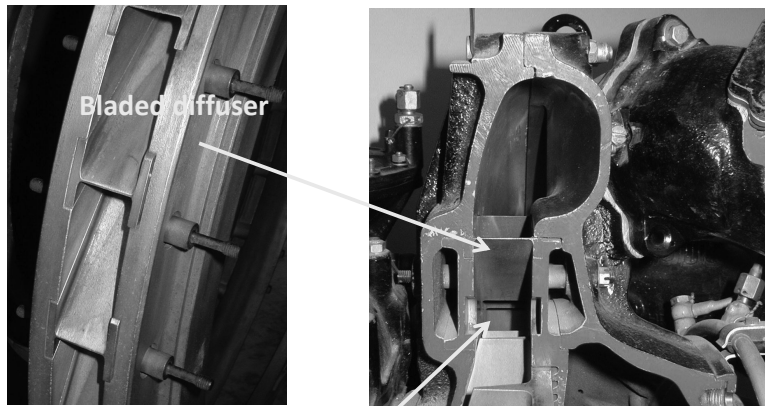
## Diffuser

- Provides transformation of air's kinetic energy to the heat and pressure energy with special shape of the channel

## Types

- Bladed
- Semi-bladed

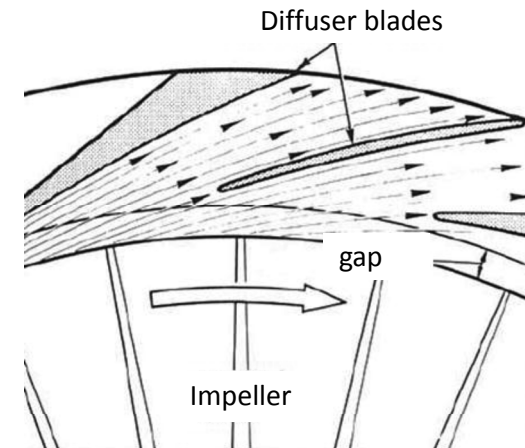
## Diffuser



Bladeless diffuser

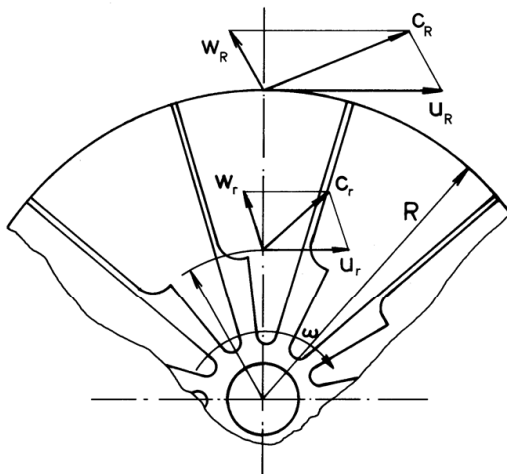
*Detail of bladed and bladeless diffuser*

## Diffuser



*Flow through the bladed diffuser [10]*

## Parameters through the compressor



*Speed triangles on inlet and outlet of the compressor*

## Parameters through the compressor

### • Velocities

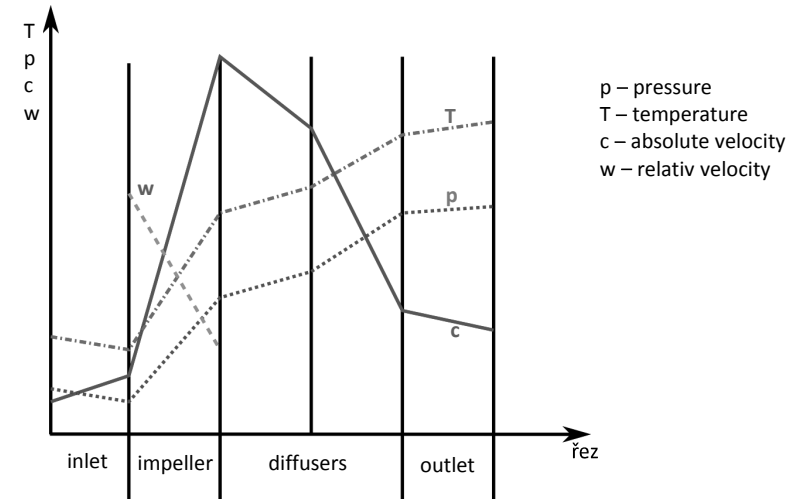
Rotating impeller have circumferential speed  $u$ , absolute velocity  $c$  and relative velocity  $w$ . Diffuser shape of the channel causes that relative velocity  $w$  is decreasing. Circumferential speed  $u$  is increasing along impeller radius toward to impeller output. By vector sum of velocities alongside the impeller radius it is obvious that the absolute velocity  $c$  have to increase toward to impeller output, where obtain it's maximum. Behind impeller there is just absolute velocity  $c$  component of velocity, because there is no rotating component. In diffuser channel absolute velocity  $c$  decreasing according to diffusivity of the channel. Further velocity decrease is caused by friction.

## Parameters through the compressor

### . Pressure and temperature

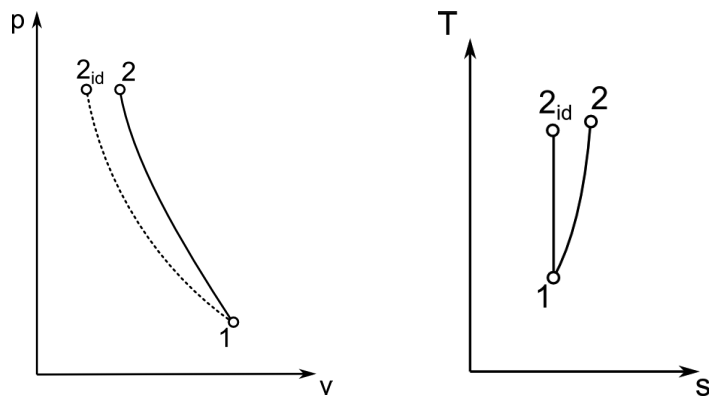
Pressure  $p$  and temperature  $T$  have similar evolution. Temperature and pressure are increasing at impeller (energy transformation). At diffuser channel is transformed kinetic energy to the heat and pressure energy so the temperature and pressure are increasing as well. This trend is continuing at outlet manifold but with smaller gradient.

## Parameters through the compressor



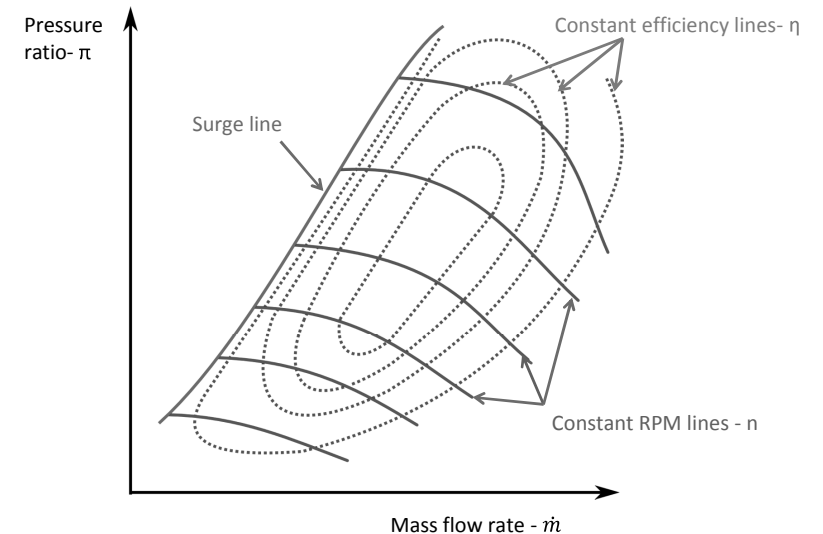
Parameters evolution through the centrifugal compressor

## Thermodynamics



$p$ - $v$  and  $T$ - $s$  diagrams of the compression

## Compressor map



Compressor map

## Reference

- [1] J. Macek; B. Suk : Spalovací motory I. - Praha 1996
- [2] L. Bartoníček: Přepřňování pístových spalovacích motorů – Liberec 2004
- [3] K. Hoffman: Regulované přepřňování vozidlových motorů. Brno, 2000.
- [4] J. Macek; V. Kliment: Spalovací turbíny, turbodmychadla a ventilátory (Přepřňování spalovacích motorů) – Praha 2003
- [5] Hiereth H., Prenninger P.: Charging the Internal Combustion Engine, Springer, Wien 2007
- [6] Bell C : Maximum Boost, Bentley Publishers, Cambridge – 1997
- [7] Baines C.N.: Fundamentals of Turbocharging, NREC, Vermont 2005

## References

- [8][http://www.dsautosolutions.ie/npics/infos/turbocharger\\_diagram.jpg\\_1.jpg](http://www.dsautosolutions.ie/npics/infos/turbocharger_diagram.jpg_1.jpg)
- [9][http://media.gm.com/content/Pages/news/us/en/2010/Nov/1109\\_gm\\_buick\\_jcr\\_content/rightpar/sectioncontainer/par/download/file.res/2011-Powertrain-4-Cylinder-Ecotec-2.0L-I4-VVT-DI-Turbo-LHU-012.jpg](http://media.gm.com/content/Pages/news/us/en/2010/Nov/1109_gm_buick_jcr_content/rightpar/sectioncontainer/par/download/file.res/2011-Powertrain-4-Cylinder-Ecotec-2.0L-I4-VVT-DI-Turbo-LHU-012.jpg)
- [10] Ott, A.: Základy teórie a konstrukce LLM I,II,III, VAAZ, Brno, 1977

**DISCUSSION...**

**...QUESTIONS**



### Poděkování

Tento projekt je spolufinancován  
Evropským sociálním fondem a státním rozpočtem České republiky

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